Electronically Sensed Hydraulic Clutch – A Review

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ABSTRACT: Electronic Clutch modification is for use by disabled drivers who prefer the control offered by a manual transmission but have difficulty operating the clutch pedal. The Device provides the driver with fingertip control of the clutch, removing the need for use of the left leg to perform gear changes. It is suitable for amputees or paraplegics, among others, especially in the case where an automatic model of a vehicle is not available.

By mounting the device on the gear stick, this modification allows the driver to operate the clutch with the left hand to make smooth gear changes. It replaces the original gear knob and incorporates a finger operated trigger lever and button. The aluminum trigger controls the clutch, with the pedal moving proportionally.

A handy switch provided with gear shift lever will operate a motor to control the functions of hydraulic cylinders by means of relay and two sensor switches. Hence fluid pressure produces in the hydraulic cylinder force the piston against spring force. This directing the clutch to engage and disengage for which fluid line will be should short circuited. This mechanism is unique in nature trouble free, cost effective and smart for its perfection.

I. INTRODUCTION

Clutch is a simple device which can be employed in any rotating drive system where it is necessary to disengage or re-engage the transmission of torque. A friction clutch is used for vehicle transmission systems. The clutch system consists of a driven and a driving member (Figure 5.25). In a transmission, the driving member is usually also the engine flywheel. The driven member is a friction disc which connects via a spline to the gearbox input shaft. The friction disc is clamped by a spring disc in the rest position, and hence drive and torque can be transmitted from driving to driven member. The spring clamping can be relaxed as required by the operator (in a vehicle transmission, via the clutch pedal); this allows the friction disc to spin freely and hence no torque is transmitted. If the spring pressure is gradually released, the clamping force can be gradually applied to the friction disc and hence torque can be progressively transferred between the driving and driven members as required. This allows the smooth take up of drive and transfer of torque between the engine and gearbox. The clutch in a vehicle transmission allows the effective disconnection of the engine and gearbox as required by the driver.

The power developed inside the engine cylinder is ultimately aimed to turn the wheels so that the motor vehicle can move on the road. The reciprocating motion of the piston turns a crankshaft rotating the flywheel thought the connecting rod. The circular motion of the crankshaft is now to be transmitted to the rear wheels. It is transmitted though the clutch, gearbox, universal joints, propeller shaft or drive shaft, differential and axles extending to the wheels.

The application of engine power to the driving wheels. The application of engine power to the driving wheels though all these parts is called power transmission. The power transmission system is usually the same on all
modern passenger cars and trucks, but its arrangement many vary according to the method of drive and type of transmission units.

Figure 1.1 Automobile Power Transmissions

Power transmissions system of an automobile shown in figure no1.1. The motion of the crank shaft is transmitted though the clutch to the gear box or transmission, which consists of a set of tears to change the speed. From gearbox, the motion is transmitted to the propeller shaft though the universal joint and then to the differential through another universal joint. Universal joint is used where the two rotating shafts are connected at an angle for power transmission. Finally, the power developed inside the cylinder is transmitted to the rear wheels though a system of transmission. The vehicle which have front wheel drive in addition o the rear wheel drives include a second set propeller shafts, universal joints, final drives and differentials for the front units.

II. TYPES OF CLUTCH & GEAR BOX

Clutch is a mechanism for transmitting rotation, which can be engaged and disengaged. Clutches are useful in devices that have two rotating shafts. In these devices, one shaft is typically driven by a motor or pulley, and the other shaft drives another device. Let us take an instance where one shaft is driven by a motor and the other drives a drill chuck. The clutch connects the two shafts so that they can either be locked together and spin at the same speed (engaged), or be decoupled and spin at different speeds (disengaged). Depending on the orientation, speeds, material, torque produced and finally the use of the whole device, different kinds of clutches are used. The clutch in itself is a mechanism, which employs different configurations and different principles in various models available. In the following lines, we have provided the different kinds of clutches that are available.

2.1 Dog Clutch

A dog clutch couples two rotating shafts or other rotating components not by friction, but by interference. Both the parts of the clutch are designed so that one pushes into the other, causing both to rotate at the same speed, so that they never slip.

2.2 Cone Clutch

Cone clutches are nothing, but frictional clutches with conical surfaces. The area of contact differs from normal frictional surfaces. The conical surface provides a taper, which means that while a given amount of actuating force brings the surfaces of the clutch into contact really slowly, the pressure on the mating surfaces increases rapidly.
2.3 Overrunning Clutch

Also known as the freewheel mechanisms, this type of clutch disengage the driveshaft from the driven shaft, when the driven shaft rotates faster than the driveshaft. An example of such a situation can be when a cyclist stops pedaling and cruises. However, in case of automobiles going down the hill, you cannot take your feet off the gas pedal, as there is no free wheel system. If you do so, the whole engine system can be damaged.

2.4 Safety Clutch

Also known as the torque limiter, this device allows a rotating shaft to "slip" or disengage when higher than normal resistance is encountered on a machine. An example of a safety clutch is the one mounted on the driving shaft of a large grass mower. If a stone or something else is encountered by the grass mower, it stops immediately and does not hamper the blades.

3.5 Centrifugal clutch

Centrifugal and semi-centrifugal clutches are employed where they need to engage only at some specific speeds. There is a rotating member on the driving shaft, which rises up as the speed of the shaft increases and engages the clutch, which then drives the driven shaft.

2.6 Hydraulic Clutch

In a hydraulic clutch system, the coupling is hydrodynamic and the shafts are not actually in contact. They work as an alternative to mechanical clutches. They are known to have common problems associated with hydraulic couplings, and are a bit unsteady in transmitting torque.

2.7 Electromagnetic Clutch

These clutches engage the theory of magnetism on to the clutch mechanisms. The ends of the driven and driving pieces are kept separate and they act as the pole pieces of a magnet. When a DC current is passed through the clutch system, the electromagnet activates and the clutch is engaged.

A clutch is a mechanical device that provides for the transmission of power (and therefore usually motion) from one component (the driving member) to another (the driven member). The opposite component of the clutch is the brake. Clutches are used whenever the ability to limit the transmission of power or motion needs to be controlled either in amount or over time (e.g., electric screwdrivers limit how much torque is transmitted through use of a clutch; clutches control whether automobiles transmit engine power to the wheels).

In the simplest application, clutches are employed in devices which have two rotating shafts (drive shaft or line shaft). In these devices, one shaft is typically attached to a motor or other power unit (the driving member) while the other shaft (the driven member) provides output power for work to be done.

In a drill for instance, one shaft is driven by a motor and the other drives a drill chuck. The clutch connects the two shafts so that they may be locked together and spin at the same speed (engaged), locked together but spinning at different speeds (slipping), or unlocked and spinning at different speeds (disengaged).
2.8 Friction clutches

Figure no 2.1 Friction clutches

Friction clutches are by far the most well-known type of clutches.

2.9 Materials

Various materials have been used for the disc friction facings, including asbestos in the past. Modern clutches typically use a compound organic resin with copper wire facing or a ceramic material. A typical coefficient of friction used on a friction disc surface is 0.35 for organic and 0.25 for ceramic. Ceramic materials are typically used in heavy applications such as trucks carrying large loads or racing, though the harder ceramic materials increase flywheel and pressure plate wear.

2.10 Push/Pull

Friction disk clutches generally are classified as push type or pull type depending on the location of the pressure plate fulcrum points. In a pull type clutch, the action of pressing the pedal pulls the release bearing, pulling on the diaphragm spring and disengaging the vehicle drive. The opposite is true with a push type, the release bearing is pushed into the clutch disengaging the vehicle drive. In this instance, the release bearing can be known as a thrust bearing (as per the image above).

2.11 Pads

Clutch pads are attached to the frictional pads, part of the clutch. They are most commonly made of rubber but have been known to be made of asbestos. Clutch pads usually last about 100,000 miles (160,000 km) depending on how vigorously the car is driven.

2.12 Dampers

In addition to the damped disc centres which reduce driveline vibration, pre-dampers may be used to reduce gear rattle at idle by changing the natural frequency of the disc. These weaker springs are compressed solely by the radial vibrations from an idling engine. They are fully compressed and no longer in use once drive is taken up by the main damper springs.

2.13 Load

Mercedes truck examples: A clamp load of 33 kN is normal for a single plate 430. The 400 Twin applications offer a clamp load of a mere 23 kN. Bursts speeds are typically around 5,000 rpm with the weakest point being the facing rivet.
2.14 Manufacturing
Modern clutch development focuses its attention on the simplification of the overall assembly and/or manufacturing method. For example drive straps are now commonly employed to transfer torque as well as lift the pressure plate upon disengagement of vehicle drive. With regards to the manufacture of diaphragm springs, heat treatment is crucial. Laser welding is becoming more common as a method of attaching the drive plate to the disc ring with the laser typically being between 2-3KW and a feed rate 1m/minute.

2.15 Multi plate clutch
This type of clutch has several driving members interleaved or "stacked" with several driven members. It is used in race cars including F1, World Rally and even most club racing, motorcycles, automatic transmissions and in some diesel locomotives with mechanical transmissions. It is also used in some electronically controlled all-wheel drive systems.

A wet clutch is immersed in a cooling lubricating fluid which also keeps the surfaces clean and gives smoother performance and longer life. Wet clutches, however, tend to lose some energy to the liquid. Since the surfaces of a wet clutch can be slippery (as with a motorcycle clutch bathed in engine oil), stacking multiple clutch discs can compensate for the lower coefficient of friction and so eliminate slippage under power when fully engaged.

The Hele-Shaw clutch was a wet clutch that relied entirely on viscous effects, rather than on friction. A dry clutch, as the name implies, is not bathed in fluid and should be, literally, dry.

2.16 Centrifugal Clutch
A centrifugal clutch is used in some vehicles (e.g., Mopeds) and also in other applications where the speed of the engine defines the state of the clutch, for example, in a chainsaw. This clutch system employs centrifugal force to automatically engage the clutch when the engine rpm rises above a threshold and to automatically disengage the clutch when the engine rpm falls low enough. The system involves a clutch shoe or shoes attached to the driven shaft, rotating inside a clutch bell attached to the output shaft. The shoe(s) are held inwards by springs until centrifugal force overcomes the spring tension and the shoe(s) make contact with the bell, driving the output. In the case of a chainsaw this allows the chain to remain stationary whilst the engine is idling; once the throttle is pressed and the engine speed rises, the centrifugal clutch engages and the cutting chain moves.

2.17 Cone clutch
Distinguished by conical friction surfaces. The cone's taper means that a given amount of movement of the actuator makes the surfaces approach (or recede) much more slowly than in a disc clutch. As well, a given amount of actuating force created more pressure on the mating surfaces. The advantage of cone clutch is that the normal force acting on contact surface is larger than the axial force.

2.18 Torque limiter
Also known as a slip clutch or safety clutch, this device allows a rotating shaft to slip when higher than normal resistance is encountered on a machine. An example of a safety clutch is the one mounted on the driving shaft of a large grass mower. The clutch will yield if the blades hit a rock, stump, or other immobile object. Motor-driven mechanical calculators had these between the drive motor and gear train, to limit damage when the mechanism jammed, as motors used in such calculators had high stall torque and were capable of causing damage to the mechanism if torque wasn't limited. Carefully-designed types disengage, but continue to transmit torque, in such tools as controlled-torque screwdrivers.
III. TYPES OF GEARBOXES

3.1 Types of gears:

![Figure 3.1 Types of gears](image1)

- **Spur**: Teeth parallel to axis, used on sliding mesh.
- **Helical**: Teeth inclined to axis to form helix.
- **Double helical**: Two sets of opposing helical teeth.
- **Epicyclic or planetary**: Spur or helical gears rotating about centers which are not stationary.

3.2 Gear ratio (single gear train)

The gear ratio, or velocity ratio, between a pair of gear wheels is in inverse ratio to the number of teeth on each.

\[
\frac{N_B}{N_A} = \frac{D_A}{D_B} = \frac{n_A}{n_B} \\
N_B = N_A \left( \frac{n_A}{n_B} \right)
\]

Where:

- \(N_A\): rev per min of gear A,
- \(n_A\): number of teeth on A
- \(N_B\): rev per min of gear B,
- \(n_B\): number of teeth on B
- \(D_A\): Diameter of gear A
- \(D_B\): Diameter of gear B

3.3 Power, Speed and Torque:

The power transmitted by a shaft is directly proportional to the speed of revolution and the torque acting on it.

\[
\text{Power [kW]} = 2\pi N T / (60 \times 1000) \text{ [N.m/s]}
\]
N = Speed in r.p.m  
T = Torque transmission N.m  

Then  
\[ T_A N_A = T_B N_B \]  
For a given power, therefore, the torque is inversely proportional to the speed of revolution and if the r/min is reduced the torque will be increased in the same ratio (assuming 100% gear efficiency).  
\[ \frac{T_B}{T_A} = \frac{n_B}{n_A} \]  
(4)

Where:
T_A = torque transmitted by A  
T_B = torque transmitted by B  

Velocity or gear ratio \((i_g)\) = number of teeth on driven gear/number of teeth on driver gear.  
\[ T_B = T_A \left(\frac{n_B}{n_A}\right) = \frac{T_A}{i_g} \]

3.4 Compound gear train  
If the number of teeth on each wheel is known, the relationship between the speed of wheels A and D can be determined as follows  
For wheels A and B:  
\[ \frac{N_B}{N_A} = \frac{n_A}{n_B}, \text{ i.e. } N_B = N_A \left(\frac{n_A}{n_B}\right) \]  
(5)  
Wheel B and C are fixed on the same shaft, so  
\[ N_C = N_B \]  
(6)  
For wheels C and D:  
\[ \frac{N_D}{N_C} = \frac{n_C}{n_D}, \text{ i.e. } N_D = N_C \left(\frac{n_C}{n_D}\right) \]  
(7)  
Substituting \(N_C = N_B = N_A \left(\frac{n_A}{n_B}\right)\) from above, we get  
\[ N_D = N_A \left(\frac{n_A}{n_B}\right) \left(\frac{n_C}{n_D}\right) \]  
(8)  

By inspection of the layout of the figure, it will be observed that wheels A and C are driver gears while B and D are driven gears. Hence, from the above equation \(n_A\) is number of wheel A \(n_B\) is number of teeth of wheel B \(n_C\) is number of teeth of wheel C \(n_D\) number of teeth of wheel D and all gears equivalent no of teeth of in this system.

Figure no 3.3 Compound Gear Train

Velocity or gear ratio \((i_g)\) = product of teeth on driven gears/ product of teeth on driver gears

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3.5 Types of Drives and gearboxes

There are many types of the car drives, usually classified according to number of driving axles (4x2, 4x4, 4WD, AWD) and each type has a different gearing arrangement. Also, gearbox (transmission) has different types (sliding-mesh, constant-mesh, synchro-mesh) some of them are old-fashion and had been replaced, and some are in use in modern cars.

3.6 Sliding-mesh gearbox:

The sliding gearbox was popular on cars up to about 1930, but it is rarely used. The basic layout of a 4-speed and reverse gearbox is shown in the figure. The various spur-type gears are mounted on three shafts.

1. Primary shaft (alternative names – clutch or first motion shaft)
2. Lay shaft (countershaft)
3. Main shaft (third motion shaft).
4. Primary shaft

This shaft transmits the drive from the clutch to the gearbox. At the end, the shaft is supported by a spigot bearing positioned close to the splines on to which the clutch driven plate is connected. The main load on this shaft is taken by a bearing; normally a sealed radial ball type, positioned close to an input gear called a constant mesh pinion. The gear is so named because it is always in mesh with a larger gear, a constant mesh wheel that I part of the lay shaft gear cluster. Note that a small driving gear is called a pinion and a large gear a wheel.

Lay shaft

This shaft, which is normally fixed to the gearbox casing, supports the various-sized driving pinions of the lay shaft gear cluster.
Mainshaft
This splined output shaft carries spur gearwheels that slide along the shaft to engage with the appropriate lay shaft gears. At the ‘front’ end, the main shaft is supported by a spigot bearing situated in the centre of the constant mesh pinion. A heavy duty radial ball bearing is fitted at the other end to take the force of the gears as the attempt to move apart.

3.7 Gear positions
Neutral
All main shaft gearwheels are positioned so that they do not touch the layshaft gears. A drive is taken to the layshaft, but the mainshaft will not be turned in neutral position.

First gear
The first-speed gearwheel A on the mainshaft is slid backwards to engage with pinion B on the layshaft; all other gears are positioned in neutral. In this gear, the reduction in speed that occurs as the drive passes through the constant-mesh gears, E and F, is reduced further by the first-speed gears, A and B.

The gear ratio (also called the movement ratio or velocity ratio) is given by:

\[
\text{Ratio} = \frac{\text{Driven}}{\text{driver}} \times \frac{\text{driven}}{\text{driver}} = \frac{F}{E} \times \frac{A}{B}
\]

\(ig_1 = \frac{F}{E} \times \frac{A}{B}\)  \(1\)

\[
\text{output} = \frac{\text{input}}{i_g}
\]

\(N_{\text{output}} = \frac{N_{\text{input}}}{i_{g1}}\)

\(T_{\text{output}} = T_{\text{input}} \times i_{g1} \times \eta_{g1}\)

Second gear
The second-speed gearwheel C is slid forward to engage with the layshaft gear D; all the other gear are set in the non-driving position.

\(ig_2 = \frac{F}{E} \times \frac{C}{D}\)  \(2\)

Third gear
In this gear position, gearwheel G is slid in to mesh with gear H.

\(ig_3 = \frac{F}{E} \times \frac{H}{G}\)  \(3\)

Figure no 3.4 Lay Shafts in 3rd gear

Top gear
In this layout, fourth gear is a direct drive; namely a gear that gives a ratio 1:1. It is obtained by sliding gear G to engage its dog teeth with the corresponding teeth formed on the end of the constant mesh pinion E. Engagement of the dog clutch locks the primary to the main shaft and this gives a ‘straight-through’ drive.
Reverse gear

Sliding a reverse gear between any two gears on the layshaft and main shaft is the method used to change the direction of rotation of the output shaft.

The simplest arrangement uses a single reverse gear, which is mounted on a short shaft. This shaft is positioned so that the reverse can slide and mesh with the two first-speed gears as shown in the figure. The gear ratio is

\[ i_{gr} = \frac{\text{Driven}}{\text{Driver}} \times \frac{\text{Driven}}{\text{Driver}} \times \frac{\text{Driven}}{\text{Driver}} \]

\[ = \frac{F}{E} \times \frac{J}{B} \times \frac{A}{J} \]

\[ = \frac{F}{E} \times \frac{A}{B} \]  

(4)

This is the same ratio as for first gear, and irrespectively of the size of gear J, it will be seen that the ratio always remains the same. For this reason it is called an idler – it changes the direction, but does not alter the ratio.

With the idler arrangement, some drivers persistently slip the clutch to maintain a low reversing speed. Excessive clutch wear resulting from this practice is minimized when the reverse ratio is set lower than first gear. This achieved by using a reverse gear arrangement as shown in the figure.

Instead of single idler, the compound reverse gear has two gear pinions joined together. The reverse shaft is positioned so that the reverse pinions are able to mesh simultaneously with the appropriate lay shaft and main shaft gears. Reverse gear mechanism arrangement use for various gear position and selecting mechanism in this method. This shaft is positioned so that the reverse can slide and mesh with the two first-speed gears.

Gear Changing

When one gear is moved to engage with another gear noise will result if the peripheral (outside) speeds are not the same to avoid this, the driver of the vehicle having a sliding-mesh gearbox performs an operation called double declutching.

Select mechanism

A fork of the type shown in figure is used to slide a gearwheel along the main shaft in order to select the appropriate gear. It is mounted on its own rod and links the driver’s gear stick to the sliding gearbox. Every gearbox must be fitted with the following:

Selector detent
Holds the gears and selectors in position and so prevent gear engagement or disengagement due to vibration. The figure shows a typical arrangement suitable for a layout having the selector fork locked to the rod.

**Interlock mechanism**

Prevents two gears engaging simultaneously; if this occurs the gearbox will lock up and shaft rotation will be impossible. Although the interlock device takes a number of different forms, the arrangement shown in the figure is one of the most common.

The interlock mechanism of used for driver change the gear center detent plug may be lose or even missing. To correct, inspect the various components of the gear shift mechanism and replace the defective parts.

Initial movement of the selector a sleeve carries the hub towards the gear and allows the cones adjusts the speed of the gearwheel to suit the hub and main shaft.

![Inter Lock Mechanism](image)

**Figure no 3.6 Inter Lock Mechanism**

![Ball and plunger type of interlocking mechanism prevents two gears engaging at the same time](image)

**Figure no 3.7 Ball and plunger type of interlocking mechanism prevents two gears engaging at the same time**

**Power take-off arrangement**
In addition to the mechanism use for driving a vehicle along a road, a power supply is often required for operating external items of auxiliary equipment. A light truck having a tipping mechanism is one example, but the most varied application of power take-off units is associated with specialized off-road vehicles.

The figure shows a typical power take-off arrangement that is driven from the gearbox lay shaft.

**Disadvantages of the sliding mesh**

Although the mechanical efficiency of the sliding mesh gearbox was high, it suffered from two great disadvantages:

1. Gear noise due to the type of gear.
2. The difficulty of obtaining a smooth, quiet and quick change of gear without the great skill and judgment.

**3.8 Constant-mesh gearbox**

![Fig no 3.8 Constant Mesh Gearbox](image)

The main feature is the use of the stronger helical of double helical gears which lead to quieter operation. In this design, the main shaft pinions revolves freely on bushes or needle-roller bearings and are all in constant engagement with the corresponding lay shaft wheels. The gear operation is obtained by locking the respective gear to the main shaft by means of a dog clutch. The layout of the box is shown in the figure.

With this arrangement the quieter-running helical gears can be employed, and during gear changing the noise and wear are reduced by the simultaneous engagement of all the dogs instead of only a pair of gear teeth as on the sliding-mesh gearbox.

With single helical pinions (double helical is economically impractical), the driving loads on the teeth cause an axial thrust which must be resisted by thrust washers, or shoulders, on the main shaft.
3.9 Constant-load synchro-mesh

The figure shows unite main details of a constant-load synchro-mesh, with the exception that a cone clutch is fitted between the dog and gear members. The initial movement of the selector a sleeve carries the hub towards the gear and allows the cones adjusts the speed of the gearwheel to suit the hub and main shaft.

3.10 Baulk ring synchro-mesh

This system is designed to overcome the main disadvantage of the earlier design- noise or crashing of the gears due to a quick change, by adding baulking ring to do the job as shown in the figure.

3.11 Additional gear ratios

Commercial vehicles having a relatively low power/weight ratio, and operating under unloaded to fully loaded conditions, require additional gears for efficient operation. The main feature is the use of the stronger helical of double helical gears which lead to quieter operation. In this design, the main shaft pinions revolves freely on bushes or needle-roller bearings and are all in constant engagement with the corresponding lay shaft wheels.

IV. DC MOTOR

A DC motor is an electric motor that runs on direct current (DC) electricity. DC motors were used to run machinery, often eliminating the need for a local steam engine or internal combustion engine. DC motors can operate directly from rechargeable batteries, providing the motive power for the first electric vehicles. Today DC motors are still found in applications as small as toys and disk drives, or in large sizes to operate steel rolling mills and paper machines. Modern DC motors are nearly always operated in conjunction with power electronic devices.

Two important performance parameters of DC motors are the Motor constants, Kv and Km. Two types component motors i) Brush ii) Brushless

4.1 Type of DC motor

There are three types of connections used for DC electric motors: series, shunt and compound. These types of connections configure how the motor's field and armature windings are connected together. The type of connection is significant because it determines the characteristics of the motor and is selected for speed/torque requirements of the load.

1. Series connection
2. Shunt connection
3. Compound Connection
4. Relay
V. FEATURES OF 89c51 MICRO CONTROLLERS

1. Compatible with MCS-51TM Products Equivalent to 8051.
2. 4k Bytes of Reprogrammable Flash Memory (Endurance: 1000 write/Erase Cycles)
3. Maximum speed of operation: 24 MHZ Can be operated at a maximum frequency of 24 MHz
4. Three-level Program Memory lock Three levels of program memory lock for software security
5. 128X 8 – Bit internal RAM 128 Bytes of RAM for temporary data storage
6. 32 programmable I/O Lines Four PORTS each having 8 lines, can be used as input or output
7. Two 16-Bit Timer/Counters Two Timer/counters for performing timing or counting operations
8. Six Interrupt Sources Two external interrupt for communicating with outside world & other Internal interrupts are for software communication
9. Programmable Serial Channel One serial communications channel for communicating with other Devices like computer etc.
10. Low power idle and Power down modes

5.1 Advantages of Micro-controller:
1. The number of discrete components required is less.
2. It is to program
3. Reliability is high
4. Easy to modify the program
5. Less complexity is hardware circuit
6. All MCUs have on-chip resources to achieve a higher level of integration and reliability at a lower cost
7. Built-in resources increase reliability because they do not require any External circuitry to be working for the resources to function
8. They are pre-tested by the manufacturer and conserve board space by Integrating the circuitry in to the MCU.
9. Microprocessors is that these are general purpose electronic processing Devices which can be programmed to executed a number of tasks.

6.1 Schematic view of experimental set-up
VI. CLUTCH MASTER CYLINDER

The detailed construction of the master cylinder had been shown in figure. In engaged condition when the clutch fork in the released position, the push rod rests against its stop due to pedals return spring. Also the pressure of master cylinder spring keeps the plunger in its back position. The flange at the end of the valve shank contacts the spring retainer.

CLUTCH SLAVE CYLINDER

Further movement of the plunger displaces fluid through the pipeline to the slave cylinder and disengage the clutch. The construction of the slave cylinder is made clear by means of fig. The return spring in the slave cylinder maintains some pressure on the release fork so that the thrust bearing is always in contact with the release levers.

STEPPER MOTOR

Stepper motor is a machine, which converts electrical energy in to mechanically energy. The motor and the gear box unit is usually located in nearby (master cylinder) hydraulic circuit.

HANDY SWITCH

A handy switch is provides with the gear shift lever. And it is connected to the relay unit. The relay unit controls the motor.

6.2 Pictorial view of experimental setup

6.5 Specifications of project

Table No 5.1 Specifications of selected system

<table>
<thead>
<tr>
<th>System</th>
<th>Selected type</th>
</tr>
</thead>
<tbody>
<tr>
<td>Clutch</td>
<td>Single plate frictional clutch</td>
</tr>
<tr>
<td>Gearbox</td>
<td>Synchromesh gearbox</td>
</tr>
<tr>
<td>Clutching motor</td>
<td>Half stepped DC motor</td>
</tr>
<tr>
<td>Input</td>
<td>DC motor with step down transformer</td>
</tr>
<tr>
<td>Control</td>
<td>Relay circuit</td>
</tr>
<tr>
<td>Actuation</td>
<td>Limit switch</td>
</tr>
</tbody>
</table>
Characteristics of system selected Clutch:
Single plate frictional clutch.

Gearbox:
Synchromesh gearbox.

Clutching motor:
- Half stepped DC motor,
- Capacity 12v dc,
- Power of motor 200watts,
- Speed 300rpm.

Transformer:
Step down transformer 1230/12v ac

Input DC motor:
- Capacity 12v dc
- Power of the motor 100watts
- Speed 600 rpm

Limit switch:
Limit switch – Normally closed.

Fabrication of clutching system
The following steps are followed for fabrication of clutching system
1. The design layout of the clutching system is to be done first.
2. Each system is selected based on the specification and need.
3. First, the clutch plate and clutch lever need to be modified to accommodate the motor and limit switch circuit.
4. The limit switch is then connected with the relay circuit which is programmed to control the motor.
5. The gearbox is now aligned with clutching system.
6. Now the entire system is assembled together.

Objective
1. To make clutching system easier.
2. To aid physically challenged person.
3. For smooth operation of clutch.
4. To reduce manpower.
5. Objective of the project is easy to change gear
6. Eliminate the clutch pedal
7. Understanding of electronic sensor system.
8. This project fix in all type of vehicles
9. This project is less cost.
10. This project is based on electronic sensor system.
VII. WORKING PRINCIPLE

7.1 Disengagement of clutch
While the driver changing the gear, he has to switch on the handy to energies the relay. So that, the energized relay operates the motor. The motor which in turn pulls the lever attached to the pushrod of the master cylinder.

The fluid comes out from the master cylinder reaches the slave cylinder under certain pressure. The pressurized fluid pushed the slave cylinder pushrod which is connected with the clutch release fork lever. So, the disengagement of clutch takes place

While press the button microcontroller energized the transformer. So stepper motor moving the clockwise direction. Master cylinder moving the forward direction and also slave cylinder moving the forward movement. So this time disengagement of the clutch.

7.2 Engagement of clutch
After changing the gear the driver has to switch off the handy switch. So that the relay will change its polarity. Then the motor rotates in opposite direction. Mean while, clutch release fork releases by means of spring force.

The fluid return back Slave cylinder to Master Cylinder. So the engagement of takes place. Cylinder push rod to control the motor ON/OFF condition.

Changing the gear release the button movement, so slave cylinder and master cylinder return back to original position. So oil is return back to the cylinder. Stepper motor is rotate reverse direction of this condition.

7.3 Applications
Light duty vehicle likes Car, Jeep, Van
Heavy duty vehicles are Bus, Lorry, Trucks

7.4 Advantages
- Simple in design
- More effective
- Smooth in operation
- Required less manpower
- Comfort driving
- Easy maintenance

VIII. CONCLUSION

This project is an innovative concept. It is a new dimension in the transmission system of a car lots of inputs are also got from the car specialists and academicians for its improvements. The concept can be transformed to a real time fitment on further development.

By implementing our project in vehicles, it’s very useful for physically unable persons. It will be very convenient for them to drive the vehicle and also ease to change the gears. By taking as a base, analyzing it, we can also control brake, accelerator by electronic means.

Using the 89c51 microcontroller we could make the relay circuit for first and reverse gear operation. And for other gears we could use speed control circuit. Here we could use stepper motor as the actuator which may get the signal from the microcontroller. Then the stepper motor may actuate the master cylinder and slave cylinder.

Using 8051 microcontroller also run this program. But 89c51 is the advanced microcontroller system for recent days. And compare to 8051 microcontroller system 89c51 microcontroller ease programmed for this condition. So recent
day’s 89c51 microcontroller used in various electronics conditions. This program also run the stepper motor is very smoothly.

**APPENDIX**

**Micro controller program:**

```assembly
; swatch
; ---------------  ;date
; p1.2=1  switch not press,0=press
; p1.7= relay output
; 30h =time count value

org 0000h
nop
ajmp begin
org 000bh
ajmp int
org 0050h
begin:  mov sp,#60h
mov tcon,#00h
clr p1.7
mov ie,#00h
mov p0,#00h
mov p2,#00h
mov 41h,#77h
mov 30h,#00h
mov th0,#06h
mov psw,#00h
mov r0,#04h
mov r1,#0ah
mov r2,#64h
setb p1.2
setb p1.3
setb p1.4
setb p1.5
setb p1.6
mov tmod,#02h
mov tcon,#02h
mov ie,#00h

keyread:  mov a,41h
          cjne a,#77h,krdend   setb p1.2
          jb p1.2,krdend
          lcall dly
          setb p1.2
          jb p1.2,krdend
          mov 41h,#55h
krdend:  ljmp keyread
```
dly:     push psw ;delay
        mov psw,#00h
        mov r6,#10h

dly1:   djnz r6,dly1 ;r5 dly counter
        pop psw
        ret

or 0400h

int:    push psw
        push acc
        push dpl
        push dph
        mov psw,#00h
        djnz r0,in1
        mov r0,#04h
        djnz r1,in1
        mov r1,#0ah
        djnz r2,in1
        mov r2,#64h
        ljmp intnest

in1:    ljmp intend

intnest: mov a,41h
        cjne a,#55h,int2
        setb p1.7
        mov a,30h
        cjne a,#02h,int3 ;calibration time =2 if you
        mov 41h,#77h ;want change it

int2:   mov 30h,#00h
        clr p1.7
        ljmp intend

int3:   setb p1.7
        inc 30h

intend: pop dph
        pop dpl
        pop acc
        pop psw
        reti
        end

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