

(An ISO 3297: 2007 Certified Organization)

Vol. 4, Special Issue 6, May 2015

Experimental Design and Fabrication of Hydraulic Rod Bending Machine

M.Santhosh¹, S.Sathish kumar², A.Selvamani³, P.Paramasivam⁴

U.G Student, Department of Mechanical Engineering, Muthayammal Engineering College, Rasipuram, Tamil Nadu,

India^{1,2,3}

Assistant Professor, Department of Mechanical Engineering, Muthayammal Engineering College, Rasipuram, Tamil

Nadu, India⁴

ABSTRACT: Now days the world is focusing into automation. Each and every work of human is reduced by a machine, but few areas like construction the usage of machines for bending rods for stirrups which are used to withstand loads in beams and columns are not done by machine because the cost of machine is high and need skilled labors to operate it. So this project is aimed to do bending operation for stirrups using hydraulics and named as hydraulic rod bending machine. The main objective of our project is to implement the hydraulic rod bending machine in the construction sites with less cost compared to the existing bending machines, and increasing the productivity of the stirrups. Hydraulic rod bending machine consist of Double acting cylinder, P-40 Direction control valve, Hoses, Motor, Pump, Rack and Pinion, Free wheel, Fixture. The rod is bent by the hydraulic cylinder piston with holding the rod in the fixture. The main advantage of our project is the square shape of the Stirrups is bent continuously without repositioning the rod in the machine.

KEYWORDS: Hydraulic Rod Bending Machine, Kinematic Mechanism, Hydraulic Operation

I.INTRODUCTION

This project is to bend the rod at the specified dimensions which is used in the building construction which called as Stirrups. Stirrup is an important reinforced element which acts as a shear reinforcement. Presently, stirrups are made manually, which suffers from many drawbacks like lack of accuracy, low productivity and resulting into severe fatigue in the operator. In manual stirrup making process, operators not only subjecting their hands to hours of repetitive motion, but in many occasions it results into several musculoskeletal disorders (MSDs).

The project is designed based on the principle of Hydraulic system. The hydraulic load has more power compare to the manual stirrup making process suffers from the many drawbacks. The construction worker not only subject their hands to hours of repetitive motion but also sometimes suffers internal injury to his body organ i.e. disorder carpal tunnel syndrome CTS, slipped disc problem etc.

II. EXPERIMENTAL SETUP

The system that we propose is that bending of square and circular stirrups. The rod is bent with the help of hydraulic force, because the power of hydraulics is very large the main aim of our project is to increase the productivity so with the help of hydraulic force we can able to bend 3-6 rods depending upon the diameter. The drawbacks in the older machines are rectified. If we place a rod for bending in the existing system we have to reposition the rod for every bend, but in our system once the rod is placed we need not reposition the rod for all bends. This is done by a special attachment of coupling the pinion wheel of a rack and pinion set to a free wheel, the function of the freewheel is to change the position of the table to 900. When the hydraulic cylinder in the forward stroke the freewheel slips and hold the fixture table at the time the rod is bent. In return stroke of the cylinder the rack rotates the pinion which is connected to the free wheel turns the table to 90^{0} .



(An ISO 3297: 2007 Certified Organization)

Vol. 4, Special Issue 6, May 2015

Hydraulic assembly and working:

The hydraulic circuit used for bending operation has been designed and shown in the Fig. 1 the components that are used in the construction of the kit are listed in the Table 1.1.

S.No	Description
1	Tank 40 liters
2	Filter
3	Vane pump
4	Electrical motor 2hp
5	Pressure relief valve
6	Pressure gauge
7	Double acting Hydraulic cylinders
8	Hoses
9	Direction control valve-4/3 (P-40)

Table 1.1 List of components



Fig.1. Direction control valve

The Hydraulic oil employed in this process is initially stored in a tank of 40 liters capacity. A filter air breather is mounted on the tank for filling the air passing into the reservoir and straining the oil while filling the reservoir. In addition a suction strainer is used to suck the oil into the system. An electrical motor of 2HP is coupled to vane pump to drive the fluid. When the motor runs, coupling that connects motor and the pump causes the rotation of the pump in turn. Depending upon the speed of the rotation of the pump, a vacuum is created which make the oil to be sucked and raised into the system. The pressure of oil depends upon the speed of the system. The pressurized oil passes through an inline check valve that prevents the return flow of the oil into the pump. This is followed by a pressure line filter to filter the oil from dust partials. A Pressure Relief Valves is connected parallel to the output line of the filter This relief valve is used to prevent the system from the effect caused by excess pressure this limit, the valve get activated and releases the excess pressure. The flow from these valves is measured using a pressure gauge mounted on the pipe line and it can be adjusted. The pressurized oil then enters the Directional Control Valve (P-40). The DCV used here is a P-40 4/3 Joystick operated spring centered DCV. At other end of DCV is connected to a Double acting cylinder. This cylinder is involved in the bending operation. When the left spool is activated, the oil flows into the piston end the of the cylinder causing the extension of the cylinder. The extension force of the cylinder is used for bending operation. Once the bending is completed, the right spool is activated. This causes the retraction of the cylinder. This process is repeated for bending stirrups controlled manually.



(An ISO 3297: 2007 Certified Organization)

Vol. 4, Special Issue 6, May 2015

III. INDIVIDUAL COMPONENT DESIGN CALCULATION

Hydraulic power:

The hydraulic oil employed in this process is initially stored in a tank 40 liters capacity. A filter air breather is mounded and the tank for filling the air passing in to the reservoir and straining the oil while filling the reservoir. An electrical motor of 2HP is coupled to vane pump to drive the fluid. When the motors run, coupling that connects

An electrical motor of 2HP is coupled to vane pump to drive the fluid. When the motors run, coupling that connects motor and the pump because the rotation of the pump turns.

• Cylinder specification :

* 1	
Bore	= 60 mm
Wall thickness	= 2.84 mm
Length	= 186 mm
Piston thickness	= 27 mm
Stroke length	= 150 mm
Pump specification	:
Max. Pressure	$= 175 \times 10^5 \text{ N/m}^2$
Force calculation	:
Area,	$A = 4.53 \times 10^{-3} m^2$
Force or load	$= \mathbf{P} \mathbf{x} \mathbf{A}$
	$=175 \times 105 \times 4.53 \times 10^{-3}$
	=79.275KN

Rack and pinion:

The need of rack and pinion in our set up is to turn the table to 90^{0} for this arrangement the pinion circumference for the pinion is calculated based on the stroke length of the piston. For 150mm stroke length the circumference.

The stroke length of the piston	= 150 mm
Stroke length the circumference of the pinion	= 600 mm
Circumference of the circle	= 600 mm
Diameter of the pinion	= 191 mm
The standard pinion of diameter is taken	= 100 mm
Groove length in the pinion	= 72 mm
No. of teeth of Pinion Z	= 37 teeth
Module (m)	= 3 mm

Dynamic beam strength calculation:

Lewis factor(y)	= 0.175 - (0.841/Z)
	= 0.175 - (0.841/37)
	= 0.1522
Velocity (V)	= Stroke Length / Time in sec
	= 150/7
	= 0.02147 m/s
Velocity (C)	= 3/(3+V)
	= 3/(3+0.02147)v
	= 0.99
Face width (b)	= 28.27 mm
Load in the pinion	= 5.9 kg
	= 59 N
Tic stress	= 56 N/mm
(P.S.G data book)	= [F] x C = 2248.19 N > 59 N (Hence design is safe)



(An ISO 3297: 2007 Certified Organization)

Vol. 4, Special Issue 6, May 2015

Parts and Assembled Diagrams:

The Frame is the main part of the machine it is the base of the machine. The hydraulic cylinder is attached to the frame. The Fixture which is the main part of the project is also designed in the modeling software and is shown in the Figure 1.2. The assembled model of the components used in the project is shown in the Figure 1.3.



Fig 1.2 Part model of Fixture



Fig 1.3 Assembled model of the machine

IV. FABRICATION PROCESS

Process Description:

The purpose of the project is to bend rods for making it as stirrups. The motor is switched on the pump is pumping the hydraulic oil. When the left spool is activated, the oil flows into the piston end the of the cylinder causing the extension of the cylinder. The extension force of the cylinder is used for bending operation. Once the bending is completed, the right spool is activated. This causes the retraction of the cylinder. The 4/3(p-40) DCV is actuated manually in the forward side this operation makes the piston in the cylinder to move forward and the process of bending the rod is done. Then the DCV is actuated backward the piston in the cylinder retracts back. When the cylinder moves back the freewheel which is coupled to the pinion rotated. The rack is connected to the piston as the piston moves the rack also slides in forward motion the freewheel slips and does not make the table to rotate. When it is in return stroke the table is rotated 900 counter clocks wise for making the machine ready for next bending operation without repositioning the rods. After the rod is bent it is removed and the next set of rods is placed for next cycle of operation.

Photos of Fabricated model:

The project is fabricated and the photo of the fabricated model and the purpose of the project are bending rods and the illustration of the working model is shown in Figure 1.4.



Fig.1.4 Fabrication Model



(An ISO 3297: 2007 Certified Organization)

Vol. 4, Special Issue 6, May 2015

V. CONCLUSION

- 1. In latest attempt a successful solution for the manual stirrup making is obtained.
- 2. By changing the fixture in the table we can obtain various sizes of the stirrups.
- 3. Instead of complicated designs the simple kinematic system is used.
- 4. In this system bending of stirrup wire can extended to its length against workstation is possible.
- 5. The rate of production is around 1000 to 1250 stirrups in 8 hours of effective working per day.
- 6. The system can be handled by any operator very easily.

7. Due to low cost and simple design this can be marketed to any of the nation.

REFERENCES

[1]. I.Muhammed Hanoof, S.Ravi Vishwanth, P.Sureshkumar, N.Saravanan, "Design and Fabrication of Hydraulic Rod Bending Machine" (International journal of research in science engineering and technology' ISSN (Online): 2319 – 8753, (2014).

- [2]. Steve Hankel Marshall Begel "Design of a Hydraulic Bending Machine" FPL-GTR-148 ISBN 978-81-775-8580-3, (2012).
- [3]. Dr.R.K.Bansal "Fluid Mechanics and Hydraulic Machines" Laxmi Publications, ISBN 978-81-318-0815-3 (2010).

[4]. Dagwa, I.M. and Ibhadode, A.O.A.. Design and manufacture of automobile disk brake pad test rig, Nigerian 'Journal of Engineering Research and Development' 4(3) 15-24 (2005).

[5]. Er.R.K.Rajput "Strength of Materials" S.Chand, ISBN 81-219-2594-0 (2006).

[6]. PSG College of Technology "PSG Design Data Book" (2012).

[7]. T.J.Prabhu "Design of Transmission Element" (2011).

[8]. V.B.Bhandari "Design of Machine Elements" McGraw Hill, ISBN 978-0-07-068179-8 (2012).

[9]. Vanalkar.A.V., Mechanism Synthesis for Stirrup Making Machine, Proceeding, 4th International Conference on Mechanical Engineering, ICME-

2001, BUET, Dhaka, BANGLADESH, pp. 271, December-2001.